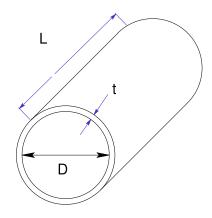
## Solution to Problem 109A



It is assumed that the flow is laminar and the fluid is Newtonian, such that the fluid displays a linear dependence of the shear stress on the shear rate. The experiment is conducted with fluid in between two concentric cylinders with one cylinder fixed. Since the radius, D/2, is much larger than the gap width,  $t \ll D/2$ , the flow is essentially Couette flow. In the assignment, the parameters are as given in table 1.

Parameter	Value	Units
diameter D	0.02	m
length L	0.2	m
thickness t	$1 \times 10^{-4}$	m
dynamic viscosity $\mu$	0.1	$\rm kg/ms$
rotational speed <b>n</b>	6000	$\operatorname{rpm}$

Table 1: Given parameters.

The torque T is defined as the force F multiplied by the radius D/2:

$$T = F\frac{D}{2} = \sigma A\frac{D}{2}$$

where the shear stress *sigma* acts in the circumferential direction for Couette flow and can be written as:

$$\sigma = \mu \frac{dU}{dy} = \mu \frac{U}{t}$$

given the linear velocity profile. The velocity U in this case can be calculated from:

$$U = n \frac{\pi D}{60} = 2\pi \ m/s \approx 6.283 \ m/s$$

Thus, the torque is given by:

$$T = \sigma A \frac{D}{2}$$
$$= \left(\mu \frac{U}{t}\right) (\pi DL) \left(\frac{D}{2}\right)$$
$$= \mu \frac{n\pi^2 D^3 L}{120t} \approx 0.790 \frac{kg m^2}{s^2}$$

The power P is defined as the torque T multiplied by the angular velocity  $\omega {:}$ 

$$P = T\omega$$
  
=  $T\frac{U}{D/2}$   
=  $\mu \frac{n^2 \pi^3 D^3 L}{3600t} \approx 496 \frac{kg m^2}{s^3}$