## An Internet Book on Fluid Dynamics

## Solution to Problem 109A



It is assumed that the flow is laminar and the fluid is Newtonian, such that the fluid displays a linear dependence of the shear stress on the shear rate. The experiment is conducted with fluid in between two concentric cylinders with one cylinder fixed. Since the radius, $D / 2$, is much larger than the gap width, $t \ll D / 2$, the flow is essentially Couette flow. In the assignment, the parameters are as given in table 1.

| Parameter | Value | Units |
| :---: | :---: | :---: |
| diameter D | 0.02 | m |
| length L | 0.2 | m |
| thickness t | $1 \times 10^{-4}$ | m |
| dynamic viscosity $\mu$ | 0.1 | $\mathrm{~kg} / \mathrm{ms}$ |
| rotational speed n | 6000 | rpm |

Table 1: Given parameters.
The torque $T$ is defined as the force $F$ multiplied by the radius $D / 2$ :

$$
T=F \frac{D}{2}=\sigma A \frac{D}{2}
$$

where the shear stress sigma acts in the circumferential direction for Couette flow and can be written as:

$$
\sigma=\mu \frac{d U}{d y}=\mu \frac{U}{t}
$$

given the linear velocity profile. The velocity $U$ in this case can be calculated from:

$$
U=n \frac{\pi D}{60}=2 \pi \mathrm{~m} / \mathrm{s} \approx 6.283 \mathrm{~m} / \mathrm{s}
$$

Thus, the torque is given by:

$$
\begin{aligned}
T & =\sigma A \frac{D}{2} \\
& =\left(\mu \frac{U}{t}\right)(\pi D L)\left(\frac{D}{2}\right) \\
& =\mu \frac{n \pi^{2} D^{3} L}{120 t} \approx 0.790 \frac{\mathrm{~kg} \mathrm{~m}}{\mathrm{~m}^{2}}
\end{aligned}
$$

The power $P$ is defined as the torque $T$ multiplied by the angular velocity $\omega$ :

$$
\begin{aligned}
P & =T \omega \\
& =T \frac{U}{D / 2} \\
& =\mu \frac{n^{2} \pi^{3} D^{3} L}{3600 t} \approx 496 \frac{\mathrm{~kg} \mathrm{~m}}{\mathrm{~m}^{3}}
\end{aligned}
$$

