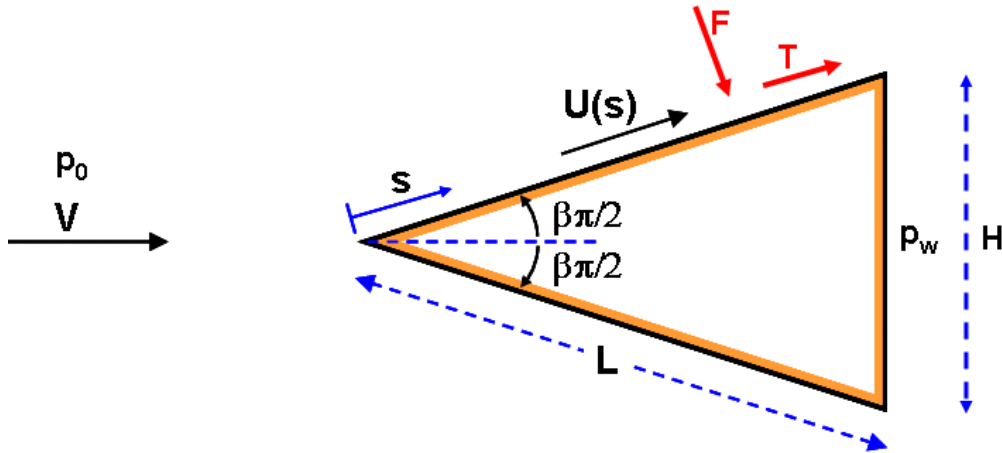


Solution to Problem 280B

The velocity outside of the boundary layer on the side of the wedge, $U(s)$ (where s is a coordinate measured along the side from the vertex of the wedge), is given by

$$U(s) = C s^m$$

From the potential flow solution past a wedge we know that $m = \beta/(2 - \beta)$. Also we are given that the velocity at the rear corner, $U(L)$, is equal to V ; therefore $C = V/L^m$.



As shown in the figure above, we will denote the pressure in the oncoming free stream by p_0 , the uniform pressure in the wake by p_w , the force (per unit dimension normal to the sketch) on the side due to the pressure by F and the force (per unit dimension normal to the sketch) on the side due to the shear stress by T . It follows from applying Bernoulli's equation to the incompressible, irrotational flow outside of the boundary layer that the pressure, $p(s)$, on the side is given by

$$p_0 + \frac{1}{2}\rho V^2 = p(s) + \frac{1}{2}\rho U^2$$

where $U(s)$ is the velocity on the side outside of the boundary layer. Therefore

$$p(s) = p_0 + \frac{1}{2}\rho V^2 - \frac{\rho V^2}{2L^{2m}}s^{2m}$$

Integrating this from $s = 0$ to $s = L$ gives us the force, F , acting normal to the side (per unit dimension normal to the sketch):

$$F = p_0 L + \rho V^2 L \frac{m}{(2m+1)} = p_0 L + \rho V^2 L \frac{\beta}{(2+\beta)}$$

Moreover, the force acting on the back of the wedge in the direction upstream (per unit dimension normal to the sketch) is $p_w H = p_0 H$ and therefore the form drag, D_F , or force on the wedge due to the pressures acting on its faces, is

$$D_F = 2F \sin \frac{\beta\pi}{2} - p_0 H = \frac{\beta}{(2+\beta)} \rho V^2 H$$

and the corresponding form drag coefficient is therefore

$$C_{DF} = \frac{D_F}{\frac{1}{2}\rho V^2 H} = \frac{2\beta}{(2+\beta)}$$

Now for the skin friction drag. The shear stress, τ_w , on the sides is given by

$$\frac{\tau_w}{\rho} = A(\beta) \frac{\nu^{\frac{1}{2}} U^{\frac{3}{2}}}{s^{\frac{1}{2}}}$$

and, substituting for $U(s)$, this becomes

$$\tau_w = \rho A(\beta) \nu^{\frac{1}{2}} C^{\frac{3}{2}} s^{\frac{(3m-1)}{2}}$$

Integrating this from $s = 0$ to $s = L$ gives us the force, T , acting tangential to the side (per unit dimension normal to the sketch):

$$T = \rho A(\beta) \nu^{\frac{1}{2}} V^{\frac{3}{2}} L^{\frac{1}{2}} \frac{2}{(3m+1)}$$

or

$$T = \frac{(2-\beta)}{(1+\beta)} A(\beta) \rho \nu^{\frac{1}{2}} V^{\frac{3}{2}} L^{\frac{1}{2}}$$

Consequently the drag, D_S , due to these shear forces

$$D_S = 2T \cos \frac{\beta\pi}{2} = 2 \frac{(2-\beta)}{(1+\beta)} A(\beta) \rho \nu^{\frac{1}{2}} V^{\frac{3}{2}} L^{\frac{1}{2}} \cos \frac{\beta\pi}{2}$$

and the corresponding skin friction drag coefficient, C_{DS} , is therefore

$$C_{DS} = \frac{D_S}{\frac{1}{2} \rho V^2 H} = 2 \frac{(2-\beta)}{(1+\beta)} A(\beta) \left(\frac{\nu}{VL} \right)^{\frac{1}{2}} \cot \frac{\beta\pi}{2}$$

or

$$C_{DS} = 2 \frac{(2-\beta)}{(1+\beta)} (0.332 + 0.87\beta) (Re_L)^{-\frac{1}{2}} \cot \frac{\beta\pi}{2}$$

where $Re_L = VL/\nu$ is the Reynolds number of the flow.

It follows that for a 7.5° half-angle wedge that $C_{DF} = 0.08$ and $C_{DS} = 10.87/(Re_L)^{\frac{1}{2}}$. Consequently for this angle wedge, the form drag and the skin friction drag will be equal at a Reynolds number, $Re_L = 1.85 \times 10^4$. For higher Reynolds numbers the form drag will dominate; for smaller Reynolds numbers the skin friction drag will dominate.